

## DEVELOPMENT AND EVALUATION OF A STEEL-COMPOSITE HYBRID COMPOSITE REPAIR SYSTEM

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### ABSTRACT

Composite materials are widely recognized as a resource for repairing damaged pipelines. The fibers in conventional composite repair systems typically incorporate E-glass and carbon materials. To provide greater levels of reinforcement a system was developed that incorporates steel half shells and an E-glass composite repair system. In comparison with other competing composite technologies, the hybrid system has a significant capacity to reduce strain in corroded pipeline to a level that has not been seen previously. Specifically, the hybrid system was used to reinforce a pipe sample having 75% corrosion subjected to cyclic pressure at 36% SMYS. This sample cycled 767,816 times before a leak failure developed. Furthermore, recent testing has demonstrated that the hybrid system actually places the pipeline in compression during installation. This paper will provide results on a series of specifically-designed tests to evaluate the performance of the hybrid system and the implications in relation to the service of actual pipelines.

### INTRODUCTION

The purpose of this paper is to provide information on a testing program conducted to evaluate the Composit Sleeve™ system in repairing and reinforcing damaged high pressure transmission pipelines. This repair system is a hybrid design that integrates steel half-shells combined with a water-activated urethane E-glass composite.

The approach used to evaluate the repair system is based on full-scale experimentation, where defects were machined into test samples. The performance of the system was evaluated by means of destructive testing. Much of this work is based on previous studies conducted by SES for the pipeline industry in evaluating competing composite repair technologies. Having access to data associated with these prior studies permits a direct comparison of the repair system with other composite repair technologies. Of particular interest is the level of reinforcement provided by the repair system to the damaged region of the pipe. Composite technologies that are most effective are those systems that are able to successfully reduce strain in the damaged region (i.e. corrosion and dents) to acceptable levels. As will be presented in this report, the repair system is effective in reducing strain in the corroded region of a pipeline significantly below industry norms. In turn, the 767,816 cycles to failure measured for the pressure cycle fatigue sample is greater than any composite repair system tested to date.

This paper has been organized to provide the reader with a brief background on the repair system, including a descriptive schematic of the system. A detailed discussion on the test program, including results, is presented. The primary means for evaluating the repair system is its ability to provide reinforcement to a simulated 75% deep

corrosion machined defect installed in a 12.75-inch x 0.375-inch, Grade X42 pipe sample. Sections of this paper include Background (with details on the repair of corrosion), Testing Methods and Results, Additional Testing: Methods and Results, Discussion, and Closing Comments.

### BACKGROUND

The Composit Sleeve™ system (hereafter referred to as the *hybrid steel-composite repair*) is a hybrid design that integrates steel half-shells combined with an E-glass composite having a water-activated urethane resin system. Unlike conventional composite repair systems that utilize fibers (i.e. typically E-glass and carbon as the primary means for reinforcement), the repair system relies on the stiffness of steel and adhesive bonding between the outer surface of the pipe and inner surface of the steel half shell. Most of the composite repair systems currently on the market are either in compliance or seeking compliance with standards such as ASME PCC-2, *Repair of Pressure Equipment and Piping, Part 4, Nonmetallic and Bonded Repairs*. These standards were written primarily for wet wrap systems involving the use of either saturation in the field or pre-impregnated resins. Systems involving either pre-cured coils or half shell designs are not explicitly addressed in the current standards. As a result, any evaluation of these systems must rely on assessments via full-scale testing performance and not rely explicitly on calculations to determine the minimum required reinforcing thickness.

Provided in Figure 1 is a schematic diagram showing the components of the hybrid steel-composite repair system. The variables of interest include selection of the load transfer material, thickness of the steel, selection of the bonding adhesive between the pipe and steel sleeves, and thickness of the E-glass composite material.

At the present time there is no single published methodology that can be used to determine the minimum required thickness for a composite-based reinforcing sleeve such as the hybrid steel-composite repair system; however, as a minimum, the following should be considered from a performance standpoint.

- Stresses must be reduced in the damaged section of pipe to an acceptable level.
- Stresses in the reinforcing materials of the hybrid steel-composite repair (steel, composite, and adhesive) must not exceed design stresses.
- The repair must be able to withstand both static and cyclic pressure loading.
- Long-term performance is an essential variable of interest in qualifying repair systems.

A calculator was developed that could be used to determine the thickness of the overwrapping E-glass composite material. Typically, the hybrid steel-composite repair half shell thicknesses have not been less than 0.25 inches. In comparison to the relative low stiffness of the composite material, this thickness of the steel has been adequate for all testing completed to date (i.e. steel pipe thicknesses of 0.375 inches). Screenshots for the calculator are presented in Figure 2 and Figure 3, along with the appropriate equations. The essential elements of this calculator include the following:

- Properties of the base pipe material including diameter, wall thickness, and grade.
- Dimensions of the corrosion.
- Aspects of the repair including length and adhesive lap shear strength.
- Safety factors, especially with regards to the bonding adhesive.

The calculated required thickness of the composite material is based on the assumption that the **composite material** and the **bonding adhesive** are the two structural components that reinforce the damaged section of pipe, and resist that opposing force caused by the internal pressure. For conservatism, it is assumed that these two structural members resist a force caused by the product of the internal pressure and projected area of the corrosion. In other words, if the steel associated with the corroded region of the pipe were removed, the composite and bonding adhesive would be responsible for resisting the entire resulting force associated with the projected opening of the corrosion defect. While this condition would never exist, this approach provides a conservative mechanics-based reference point on which to make calculations.

Note the following input data in Figure 2.

- Pipe: 12.75-inch x 0.375-inch, Grade X42
- Corrosion geometry: 8 inches long x 6 inches wide
- Adhesive shear strength: 2,000 psi
- Composite tensile strength: 20,000 psi (long-term design strength)
- Length of repair: 18 inches
- Adhesive service factor: 0.2 (safety factor of 5)
- Composite service factor: 0.5 (safety factor of 2 on long-term strength)

The design factors that contribute to the strength of the reinforcing system include the shear strength of the adhesive, tensile strength of the composite material, composite thickness, and the length of the repair. Assuming that material selection is not an option, the composite thickness and length of repair are the two critical variables.

As observed in Figure 3, the resulting composite thickness is 0.142 inches. If the shear strength of the adhesive,  $t_{adh}$ , is varied, the following composite thickness,  $t_c$ , values are calculated:

- $t_{adh} = 2,375$  psi     $t_c = 0.010$  inches
- $t_{adh} = 2,000$  psi     $t_c = 0.142$  inches (default configuration)
- $t_{adh} = 1,000$  psi     $t_c = 0.492$  inches
- $t_{adh} = 500$  psi     $t_c = 0.668$  inches
- $t_{adh} = 100$  psi     $t_c = 0.808$  inches

What is interesting to note in the above calculations is the significant contribution that the adhesive makes to the overall strength of the reinforcement. As noted, when the shear strength is assumed to be 2,375 psi, the need for composite reinforcement is practically non-existent. Of the three burst tests that were conducted, the test sample

with no composite reinforcement did indeed demonstrate that the adhesive-only case had adequate strength to resist the corrosion bulging and pipe expansion associated with the burst test.

## TESTING TO ADDRESS REPAIR OF CORROSION

Full-scale testing was performed to evaluate the performance of the hybrid steel-composite repair system in repairing a corroded pipeline. While analytical calculations are useful for sizing the components within a system, it is essential that performance testing be conducted to validate the overall capabilities of a given composite reinforcement system. What is lacking in the calculations is the inter-dependent relationship that exists between the components in the system. For example, the load transfer material serves to ensure that load generated by the bulging corrosion is transferred into the composite fiber material. A filler material lacking the required rigidity will fail to engage the composite material, resulting in an excessive accumulation of strain in the reinforced steel material beneath the reinforcement. The end result in having a deficient filler material is a highly-loaded corrosion region that is susceptible to premature failure in the form of leaking, especially in the presence of cyclic pressure loading.

The sections of this paper that follow provide specific details associated with the testing methods and results used to evaluate the performance of the hybrid steel-composite repair system in reinforcing a corroded pipeline.

### Testing Methods

Two types of testing were conducted. The first involved samples repaired to evaluate the increase in burst strength (i.e. limit state) of a corroded pipe section, while an additional test samples was prepared to evaluate the effects of cyclic pressure on the performance of the hybrid steel-composite repair.

The variable of interest in this study was the thickness of the E-glass water-activated urethane. An additional sample was also fabricated to determine what would happen if no composite materials were installed on the outside of the repair. The pipe test samples included a 12.75-inch x 0.375-inch, Grade X42 pipe with a corrosion region having a depth of 75% installed by machining. Figure 4 shows the schematic for this test sample. Strain gages were installed in the machined corrosion region, on the base pipe, and outside of the repair, as shown in Figure 5.

As stated previously, three types of tests were conducted, with details for the burst sample configurations listed below. The pressure cycle fatigue sample involved the *steel plus 18 wraps* configuration.

- Sample #1: Steel only
- Sample #2: Steel plus 18 wraps
- Sample #3: Steel plus 8 wraps

Figures 6 through 10 provide a series of photographs showing actual measurements made after the machining was completed. The target machining depth was 75%, although as can be seen from these photographs, the machining depth averages ranged from 74% to 78% of the pipe's nominal wall thickness. After machining, end caps were welded to each sample. The samples were sandblasted to a near white metal finish (NACE 2) prior to installation of the repair system.

In addition to the corrosion tests, results are also presented for a test designed to address the performance of the system in repairing leaks. Results are also provided for measurements taken during installation of the sleeve that actually quantified the level of compression measured during installation.

## Test Results

Results are presented in this section of the paper detailing the measurements and final performance data associated with both the burst and pressure cycle fatigue tests. In addition to the actual performance data (i.e. burst pressure and number of cycles to failure), data were also generated from the strain gage measurements in the corroded steel region beneath repair. The strain measurements are a fundamental means for evaluating the ability of a repair system to reinforce a damaged section of pipe. With Stress Engineering Services, Inc. having performed corrosion repair tests on more than 15 different systems over the past 5 years, it is possible to evaluate the relative performance of a particular system to industry norms. A comparison of this type will be provided for the hybrid steel-composite repair system.

### Burst Tests

The three burst tests were performed to determine the limit state capacity for each repair configuration. The recorded burst pressures for the three different repair configurations.

- Sample #1 (Steel only):  $P_{burst} = 3,995$  psi
- Sample #2 (Steel plus 18 wraps):  $P_{burst} = 4,389$  psi
- Sample #3 (Steel plus 8 wraps):  $P_{burst} = 4,374$  psi

Figure 11 provides photographs of the three burst tests. Note that the failures occurred in the repaired region for all three samples; however, the failures occurred at pressure levels that would be expected for a perfect, undamaged pipe.

For the 12.75-inch x 0.375-inch, Grade X42 pipe, which has a minimum tensile strength of 60,000 psi, the estimated burst pressure is 3,529 psi. Even the minimum burst pressure of 3,995 psi (Sample #1 with steel only) is 113% of this value; while the maximum burst pressure associated with Sample #2 (i.e. 4,389 psi) is 124% of this value.

Of the three burst tests that were conducted, Sample #1 provides a significant contribution to the overall study because prior to testing, one of the questions to be addressed in the current scope was to identify the level of reinforcement provide by a repair configuration having only the adhesive bonding the steel half shell to the outer pipe surface. While Samples #2 and #3 demonstrate that the composite material provides additional reinforcement, Sample #1 shows that the adhesive by itself generates a robust level of reinforcement.

While the burst pressures are essential for understanding the limit state performance of a given repair, equally if not more important, is measuring the level of strain beneath the repair. For a repair system to work properly, it must ensure that strain levels in the damaged section of pipe (i.e. corrosion in this particular study) are minimized. There is no current limitation on strain in the reinforced section specified by any of the current design codes such as ASME PCC-2; however, it is possible to quantify acceptable strain levels by considering performance of the repair in testing. Burst testing is one means of establishing performance, although a better means for quantifying the capability of a repair system for corrosion and dents is via pressure cycle fatigue testing. The section that follows, *Pressure Cycle Fatigue Tests*, provides details on the pressure cycle fatigue test that was conducted on the hybrid steel-composite repair system.

Presented in this section of the report are the strain gage results measured during burst testing of the three different repair configurations. The presentation includes plotted data, as well as tabulated data measured at pressure levels equal to 72% and 100%

SMYS (1,780 psi and 2,470 psi, respectively). Figure 12 plots hoop strain as a function of internal pressure. Results are included for the following data sets:

- Leading E-glass technology
- Base pipe (non-corroded section)
- Sample #1: Steel only
- Sample #2: Steel plus 18 wraps
- Sample #2: Steel plus 18 wraps (outside surface of repair)
- Sample #3: Steel plus 8 wraps
- Data point at 72% SMYS for Industry average for other competing E-glass systems<sup>1</sup>
- Data point at 100% SMYS for Industry average for other competing E-glass systems<sup>1</sup>

There are several observations made in viewing this plot; however, the most noteworthy point is the significant reduction in strain provided by the hybrid steel-composite repair system when compared to other E-glass technologies. The large number of fatigue cycles acquired during the pressure cycle test is testimony to the appreciable level of performance (i.e. strain reduction) provided by the hybrid steel-composite repair system.

Also included are the tabulated data shown in Figure 13. As observed at 72% SMYS the PRCI average for E-glass material was 4,497  $\mu\epsilon$ , (where  $\mu\epsilon$  is microstrain with 10,000  $\mu\epsilon$  corresponding to 1% strain). For the hybrid steel-composite repair systems that were tested, the minimum measured strain data was 1,610  $\mu\epsilon$  (steel + 8 wraps), while the maximum measured of the three data sets was 2,220  $\mu\epsilon$ . Results at the 100% SMYS pressure level demonstrated similar performance characteristics when compared to the PRCI data set, although the results for the hybrid steel-composite repair are not as non-linear as the data for the PRCI E-glass systems.

### Cyclic Pressure Fatigue Tests

For the pressure cycle fatigue tests the 12.75-inch x 0.375-inch, Grade X42 pipe having 75% corrosion was cycled from 36% to 72% SMYS until a leak developed in the corroded region beneath the repair. The hybrid steel-composite repair system achieved 767,816 cycles before a leak failure developed in the repair. Of the 11 different repair configurations tested via pressure cycle fatigue to date, the hybrid steel-composite repair system achieved the greatest number of cycles to failure. Provided below are data for other test samples.

- E-glass system: 19,411 cycles to failure (MIN)
- E-glass system: 32,848 cycles to failure
- E-glass system: 129,406 cycles to failure
- E-glass system: 140,164 cycles to failure
- E-glass system: 165,127 cycles to failure
- Carbon system (Pipe #1): 212,888 cycles to failure
- Carbon system (Pipe #2): 256,344 cycles to failure
- Carbon system (Pipe #3): 202,903 cycles to failure
- E-glass system: 259,537 cycles to failure
- Carbon system (Pipe #4): 532,776 cycles (run out, no failure)
- ComposiSleeve™ system: 767,816 cycles to failure (MAX)

It should be noted that in order to achieve this number of cycles, the welds joining the end caps to the ComposiSleeve™ test sample had to be repaired multiple times. As with the burst tests, the pressure cycle

<sup>1</sup> These data acquired as part of the long-term testing program (MATR-3-4) being co-sponsored by the Pipeline Research Council International, Inc. and 13 other composite repair systems. For additional details consult [www.compositerepairstudy.com](http://www.compositerepairstudy.com).

fatigue tested integrated strain gages installed in the corroded region beneath the repair. The strain gages measured hoop and axial strains at designated cycle periods during testing. Figure 14 plots the maximum hoop strain and hoop strain range for Gage #1 located beneath the repair up to 100,000 cycles, while Figure 15 lists hoop and axial strains at designated cycle periods. The strain range is the difference between the maximum and minimum strains.

In viewing the strain data from Figure 14 and Figure 15 there are several important observations. Note that the data plotted in Figure 14 is for Gage #1 located beneath the repair.

- The maximum measured strain beneath the repair (i.e. Gage #1) increased from 1,756  $\mu\epsilon$  at start-up to 3,318  $\mu\epsilon$  at 100,000 cycles. This is approximately a 90% increase over the designated cycle period.
- Over the 100,000 cycle recording period there was actually a decrease in the Gage #1 hoop strain range from 914  $\mu\epsilon$  at start-up to 810  $\mu\epsilon$  at 100,000 cycles. This represents a strain reduction on the order of 12%.
- The strain range in the base considering the 36% SMYS pressure differential was 460  $\mu\epsilon$ . The average hoop strain range measured during the 100,000 cycle period was 811  $\mu\epsilon$ , a value that is approximately 76% more than the nominal base pipe.
- The DOE B-curve is provided below, where  $\Delta S$  represents strain range in psi. If the maximum hoop strain of 914  $\mu\epsilon$  is considered, the corresponding design life is estimated to be 1,836,094 cycles. The actual fatigue failure data was 767,816 cycles. The likely explanation for the difference is rooted in the presence of a stress concentration factor (SCF) in the vicinity of the machined corrosion. A stress concentration of only 1.24 generates a cycle to failure condition of 776,000 cycles using the DOE B-curve (a value close to the actual fatigue life of 767,816 cycles). An SCF of this magnitude is completely consistent what could be expected for a machined corrosion region.

$$N = 2.343E15 \cdot \left[ \frac{\Delta\sigma}{0.145} \right]^{-4}$$

- It is clear from the significant number of cycles achieved during testing that the hybrid steel-composite repair system is uniquely effective in reducing strain in the reinforced region of the pipe. This statement is reinforced by the fact that the hybrid steel-composite repair system was able to increase the fatigue life beyond other composite-based systems (although one carbon data point at 532,776 cycles is a run out condition).
- To be useful from a design standpoint, the 767,816 cycles to failure must be adjusted to account for variations in fatigue data and to introduce an acceptable level of conservatism. In other words, even though the test ran for more than 750,000 cycles, it is not appropriate to assume that an actual repair can function for this many cycles. If a safety factor of 10 is employed, the number of design cycles for the hybrid steel-composite repair system is 76,781 cycles (i.e. 767,816 cycles / 10).
- From a high pressure gas pipeline standpoint pipeline applications standpoint, the 76,781 design cycles for a pressure range of 36% SMYS corresponds conservatively to a design condition of **21 years** for a *very aggressive* pressure cycle condition (3,683 cycles per year), **228 years** for a *moderately aggressive* pressure cycle condition (337 cycles per year), and **600 years** for a light pressure cycle condition (128 cycles per

year). The data from Kiefner et al<sup>2</sup> in Figure 16 was used in generating the above service life values.

### Leak Repair Testing

In addition to the burst and pressure cycle tests, repairs were made on two (2) test samples, using 12.75-inch x 0.375-inch, Grade X42 pipe material, with each having a 60% deep corrosion defect. One of these test samples had a thru-wall hole (Sample #1). Prior to burst testing, each test sample was subjected to 22,503 pressure cycles from 890 psi to 1,780 psi, or from 50% to 100% of the maximum operating pressure (MAOP). After the pressure cycles, the samples were subjected to a static burst test. Sample #1 developed a noticeable leak at approximately 4,000 psi of internal pressure. The maximum pressure reached was 4,163 psi. Sample #1 did not burst due to a leak that developed at the high pressure condition. The leak occurred in the repaired region. Sample #2 burst in the base pipe away from the repair where the maximum pressure applied to the sample was 4,033 psi. Figure 17 provides two photographs of the thru-wall defect test sample prior to repair.

### Pipe Compression During Installation

In addition to the tests results provided in this paper, a series of measurements were made to measure stresses generated in the pipe beneath the hybrid steel-composite repair reinforcement. What makes this repair system unique is not only the fact that it employs steel as part of the reinforcement, but that the pipe is actually placed into compression during the process of installation. The same strain gages that were used to make measurements during testing during a recent project focused on reinforcement vintage girth welds were also used to measure strains during installation. Provided in Figure 18 is a schematic showing the geometry for girth weld sample used to measure compressive stress in the pipe beneath the reinforcement.

In the phase of testing two samples used for measurement, with three bi-axial strain gages placed on each sample that were used for monitoring the level of compression during installation. Refer to Figure 18 for locations of the gage relative to the girth weld position. Table 1 provides a summary of stresses measured during compression phase of testing. As noted in this table, a maximum compressive hoop stress of 43,860 psi with an average compressive stress on the order of 30,000 psi.

The test data presented previously in this paper have supported the concept that this particular reinforcing system has the ability to reduce strain in the base pipe. These particular measurements provide insight as to why the system has performed well. The ability of the system to actually place the pipe into compression partially explains the basis for the low strains measured in the 75% corrosion sample during pressurization. Additionally, the extremely long fatigue life achieved with the 75% corrosion sample (i.e. 767,816 cycles) is better understood in the context of a compressive stress field that reduces the mean cyclic stress, as well as introducing the possibility that the machined corroded region was at least partially in compression during the early stages of pressure cycling.

<sup>2</sup> Kiefner J. F. et al, *Estimating Fatigue Life for Pipeline Integrity Management*, Paper No. IPC04-0167, Presented at the International Pipeline Conference, Calgary, Canada, October 4 – 8, 2008.